

Calculation

Resulting and equivalent bearing loads

For angular-contact thrust ball bearings LGN and LGF

Angular-contact thrust ball bearings are preloaded. The chart shows the resulting axial bearing load F_{ax} as a function of preload and axial operating load F_{Lax} .
For a purely axial load $F_{comb} = F_{ax}$.

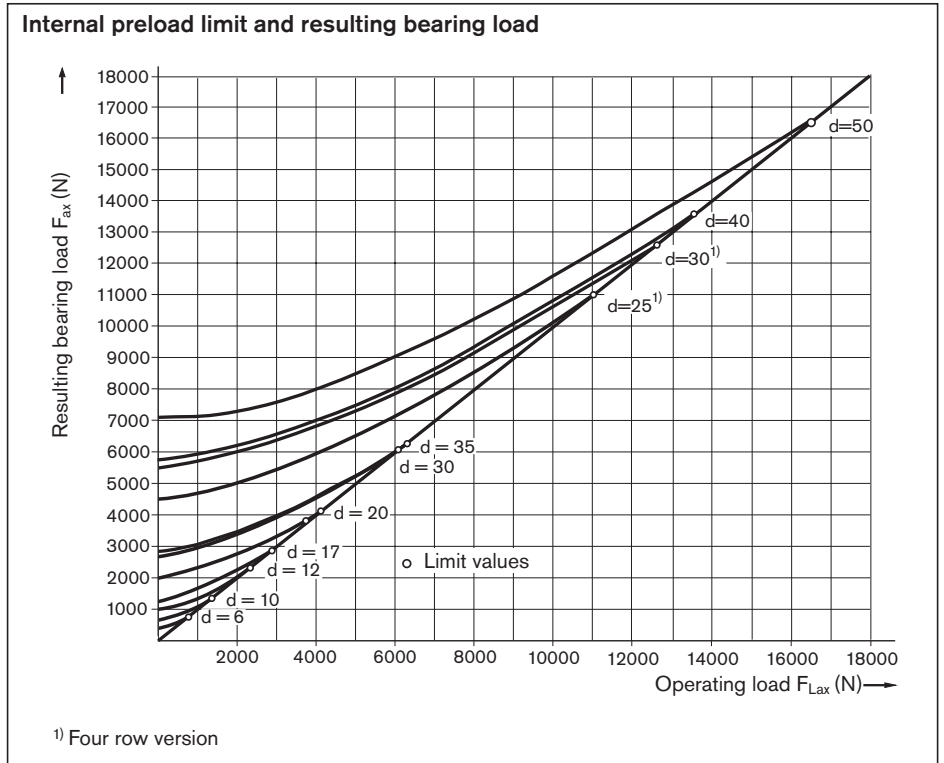
$\alpha = 60^\circ$	X	Y
$\frac{F_{ax}}{F_{rad}} \leq 2.17$	1.90	0.55
$\frac{F_{ax}}{F_{rad}} > 2.17$	0.92	1.00

α = pressure angle
 F_{ax} = resulting bearing load
 F_{Lax} = operating load
 X, Y = dimensionless factor

If the radial operating forces are not insignificant, the equivalent bearing loads are calculated according to formula 20.
 Bearings for Ball Screw Assemblies are also suitable to accommodate tipping forces.
 The moments that usually occur due to the mass and drive motion of the screw do not generally need to be included in the calculation of the equivalent bearing load.

$$F_{comb} = X \cdot F_{rad} + Y \cdot F_{ax} \quad 20$$

F_{ax} = resulting axial bearing load (N)
 F_{comb} = combined equivalent load (N)
 F_{rad} = radial bearing load (N)



Permissible static axial load for bearing series LGF

The permissible static axial load of LGF series bearings in screw-down direction is:

$$F_{0ax p} \leq \frac{C_0}{2}$$

$F_{0ax p}$ = permissible static axial bearing load (N)

The static axial load rating C_0 is stated in the Dimension Tables.

⚠ Separate technical dimensioning to determine the limit values is absolutely necessary for all attachments (e.g. pillow block units, bearing assembly, etc.)

Calculation

Resulting and equivalent bearing loads

For angular-contact thrust ball bearings LGL

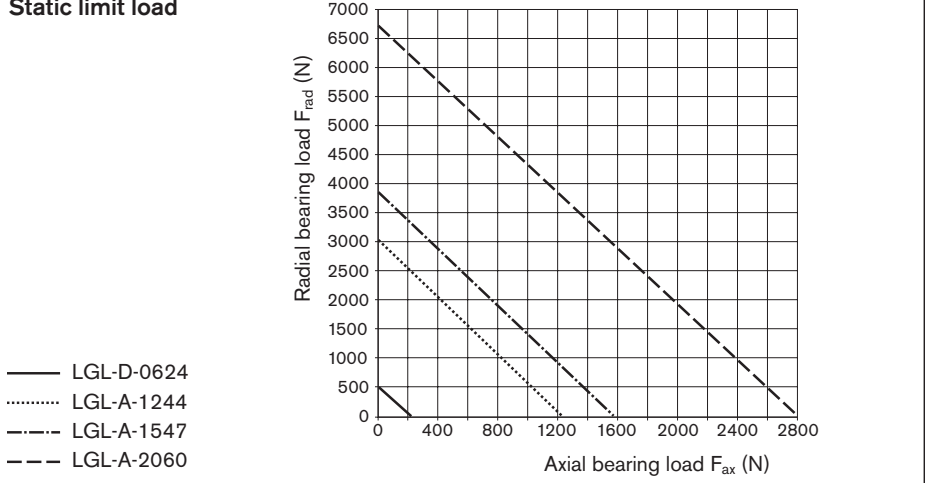
Before determining the combined equivalent load, F_{comb} , you must check the bearing size for the static limit load using the diagram. In this connection, the intersection point of the axial and radial bearing load must be below the muss boundary for a bearing to be suitable for the application.

$$F_{comb} = X \cdot F_{rad}^A + Y \cdot F_{ax}^B + Z \quad 21$$

Bearing size	X	Y	Z	A	B
LGL-D-0624	0.003	0.1300	140	1.90	1.40
LGL-A-1244	0.076	0.0460	580	1.28	1.30
LGL-A-1547	0.022	0.0110	540	1.45	1.50
LGL-A-2060	0.017	0.0082	960	1.45	1.50

- F_{ax} = axial bearing load (N)
- F_{comb} = combined equivalent load (N)
- F_{rad} = radial bearing load (N)
- X, Y, Z = calculation factors (-)
- A, B = exponents (-)

Static limit load



Average speed and average bearing load

When the bearing load varies in steps over a specific period of time **22**, calculate the dynamic equivalent bearing.

When the speed varies, use formula **23**. In these formulas q_t denotes the discrete time steps for the individual phases in %.

$$F_m = \sqrt[3]{F_{comb1}^3 \cdot \frac{|n_1|}{n_m} \cdot \frac{q_{t1}}{100} + F_{comb2}^3 \cdot \frac{|n_2|}{n_m} \cdot \frac{q_{t2}}{100} + \dots + F_{combn}^3 \cdot \frac{|n_n|}{n_m} \cdot \frac{q_{tn}}{100}} \quad 22$$

$$n_m = \frac{q_1}{100} \cdot n_1 + \frac{q_2}{100} \cdot n_2 + \dots + \frac{q_n}{100} \cdot n_n \quad 23$$

- $F_{comb1} \dots F_{combn}$ = combined equivalent axial load in phases 1 ... n (N)
- F_m = dynamic equivalent bearing load (N)
- $n_1 \dots n_n$ = speeds in phases 1 ... n (rpm)
- n_m = average speed (rpm)
- $q_{t1} \dots q_{tn}$ = discrete time steps in phases 1 ... n (%)

Service life and load safety factor

Nominal service life

The nominal service life is calculated as follows:

Attention:

Pay attention to the dynamic load rating of the nut!

$$L = \left(\frac{C}{F_m} \right)^3 \cdot 10^6 \quad 24$$

- C = dynamic bearing load rating (N)
- F_m = combined equivalent load on bearing (N)
- L = nominal service life in revolutions (-)
- L_h = nominal service life in operating hours (h)
- n_m = average speed (rpm)

$$L_h = \frac{16\,666}{n_m} \cdot \left(\frac{C}{F_m} \right)^3 \quad 25$$

Static load safety factor

The static load safety factor for machine tools should not be lower than 4.

$$S_0 = \frac{C_0}{F_{0max}} \quad 26$$

- F_{0max} = maximum static load (N)
- C_0 = static load capacity (N)
- S_0 = static load safety factor (-)

Bosch Rexroth
 Linear Motion Technology

97419 Schweinfurt / Germany

Find your local contact person here: www.boschrexroth.com/adressen

Company: _____
 Contact: _____
 E-mail: _____
 Telephone: _____

Application

New design

Revised design

Operating conditions

Discrete time step parameters			or	Dynamic cycle parameters											
Discrete time steps (%)	Speed (1/min)	Action of force x	Section	T1	T2	T3	T4	T5	T6	T7	T8	T9	T10	T11	T12
T ₁ =	n ₁ =		Path (mm)												
T ₂ =	n ₂ =		V (m/s)												
T ₃ =	n ₃ =		a (m/s ²)												
T ₄ =	n ₄ =		Time (s)												
T ₅ =	n ₅ =		Action of force x												
T ₆ =	n ₆ =														

	F1	F2	F3	F4	F5	F6
Forces (N) =						
Mass (kg) =						
Max. stroke (mm) =						

Bearing type

1. Tight

2. Tight

3. Tight

Tight

Loose

Free

Installation Position

Horizontal

Vertical

Drawing enclosed (recommended)

Delivery with bearing

Required life: _____ Operating temperature: _____ °C Up to _____ °C

Type of lubrication: _____

Short description of the application / unusual operating conditions: _____

Visit our official homepage and use the provided configurators and our dimensioning program Linear Motion Designer free of charge.